

The Aeroacoustics of Swirling Flow

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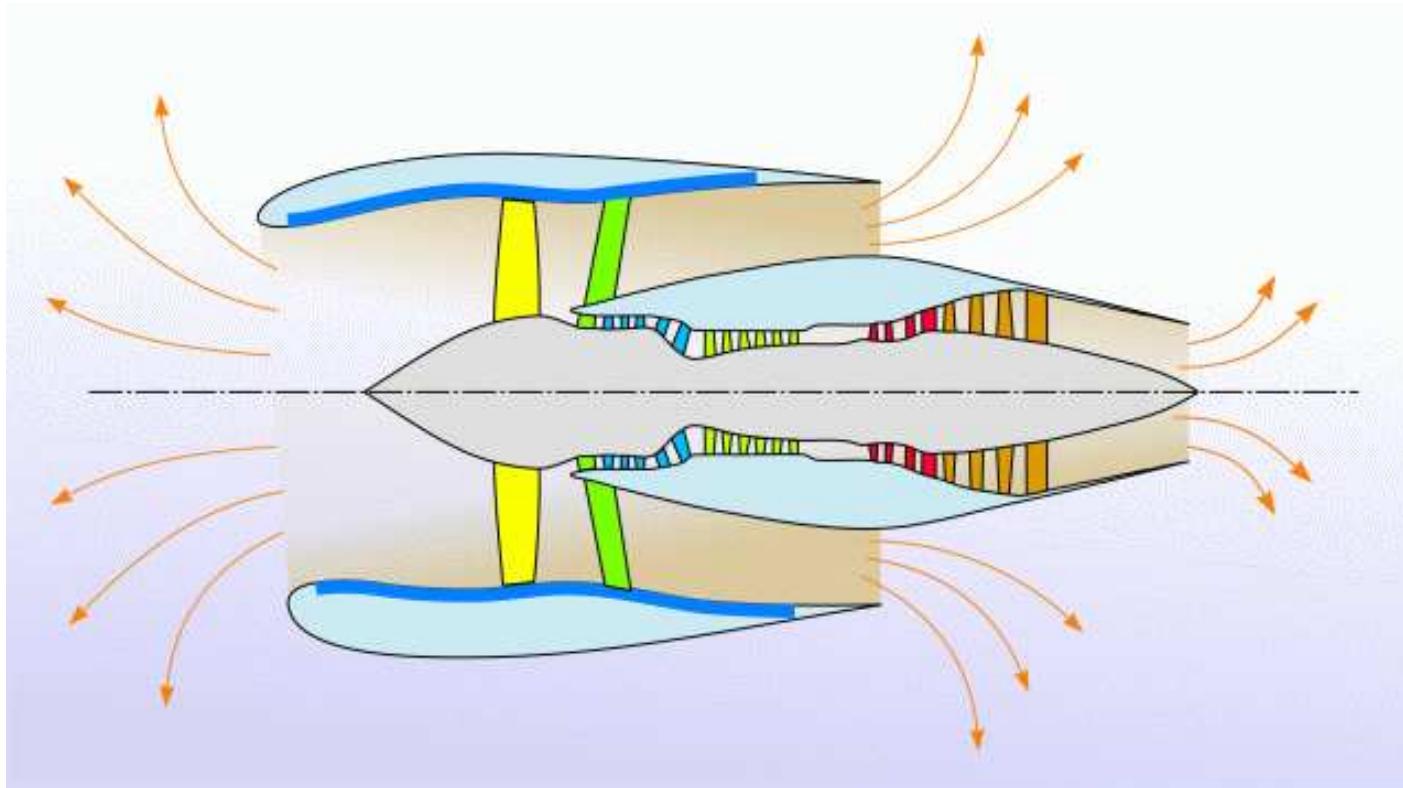
Joint work with:

- Alison Cooper, Cambridge, U. Warwick
- Chris Heaton, Cambridge
- Helene Posson, Cambridge, Airbus.

N. Peake & A.B. Parry 2012 Modern Challenges
Facing Turbomachinery Aeroacoustics.

Annual Reviews of Fluid Mechanics. 44.

Typical aeroengine

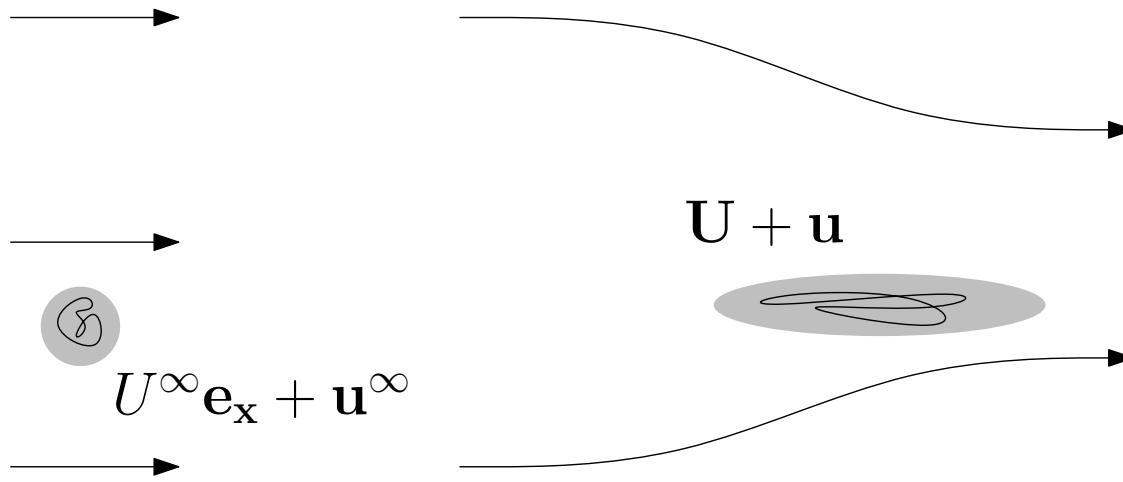


Courtesy of Sjoerd Rienstra

Plan of Talk

- Some basic properties.
- Some asymptotic results.
- Acoustic analogy in swirling flow.

Rapid Distortion Theory (RDT)



- Total velocity decomposed as $\mathbf{U}_0(x, r, \theta) + \mathbf{u}(x, r, \theta, t)$.
- Unsteady perturbation velocity decomposed as $\mathbf{u} = \mathbf{a} + \nabla\phi$.
- Unsteady pressure given by $p = -\rho_0 \frac{D_0\phi}{Dt}$.

RDT equations

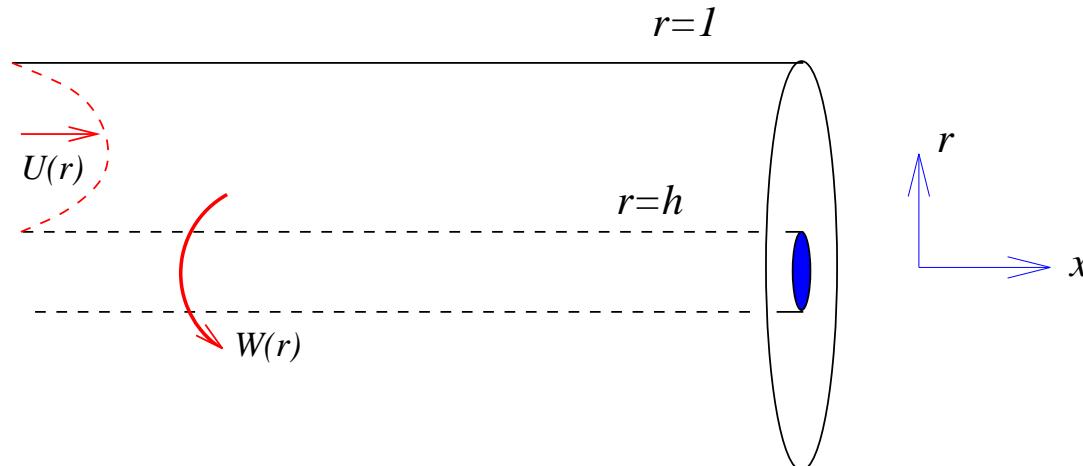
$$\frac{D\mathbf{a}}{Dt} + \mathbf{a} \cdot \nabla \mathbf{U}_0 = -\boldsymbol{\Omega}_0 \wedge \nabla \phi,$$

$$\frac{D}{Dt} \left(\frac{1}{c_0^2} \frac{D\phi}{Dt} \right) \frac{1}{\rho_0} \nabla \cdot (\rho_0 \nabla \phi) = \frac{1}{\rho_0} \nabla \cdot (\rho_0 \mathbf{a}),$$

- $\boldsymbol{\Omega}_0 = \nabla \wedge \mathbf{U}_0$ is the mean vorticity.
- $\boldsymbol{\Omega}_0 = 0$ means that the unsteady vorticity is uncoupled from the unsteady pressure. **Classical RDT**.

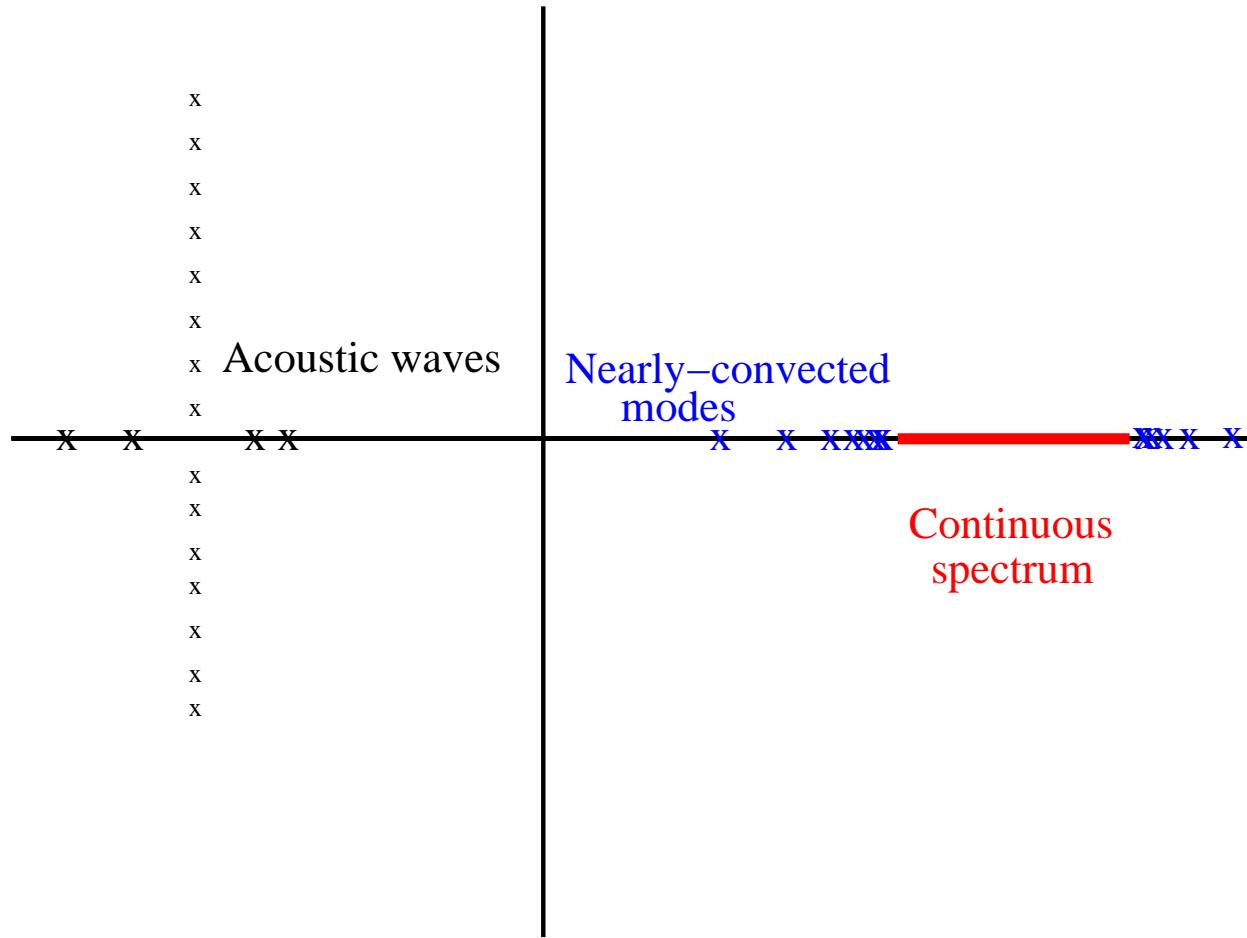
The spectrum

- Consider cylindrical duct $h \leq r \leq 1$. Maybe $h = 0$.
- Mean flow $\mathbf{U}(r) = U(r)\mathbf{e}_x + W(r)\mathbf{e}_\theta$.
- Assume all unsteady flow quantities proportional to
$$\exp(i k x + i m \theta - i \omega t)$$
- What are the allowed values of k ? And the corresponding modes?

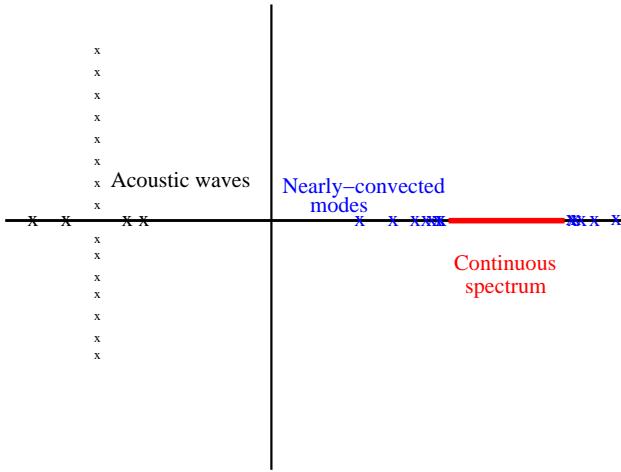


Possible axial spectrum

Eigensolutions in cylindrical duct with rigid walls.



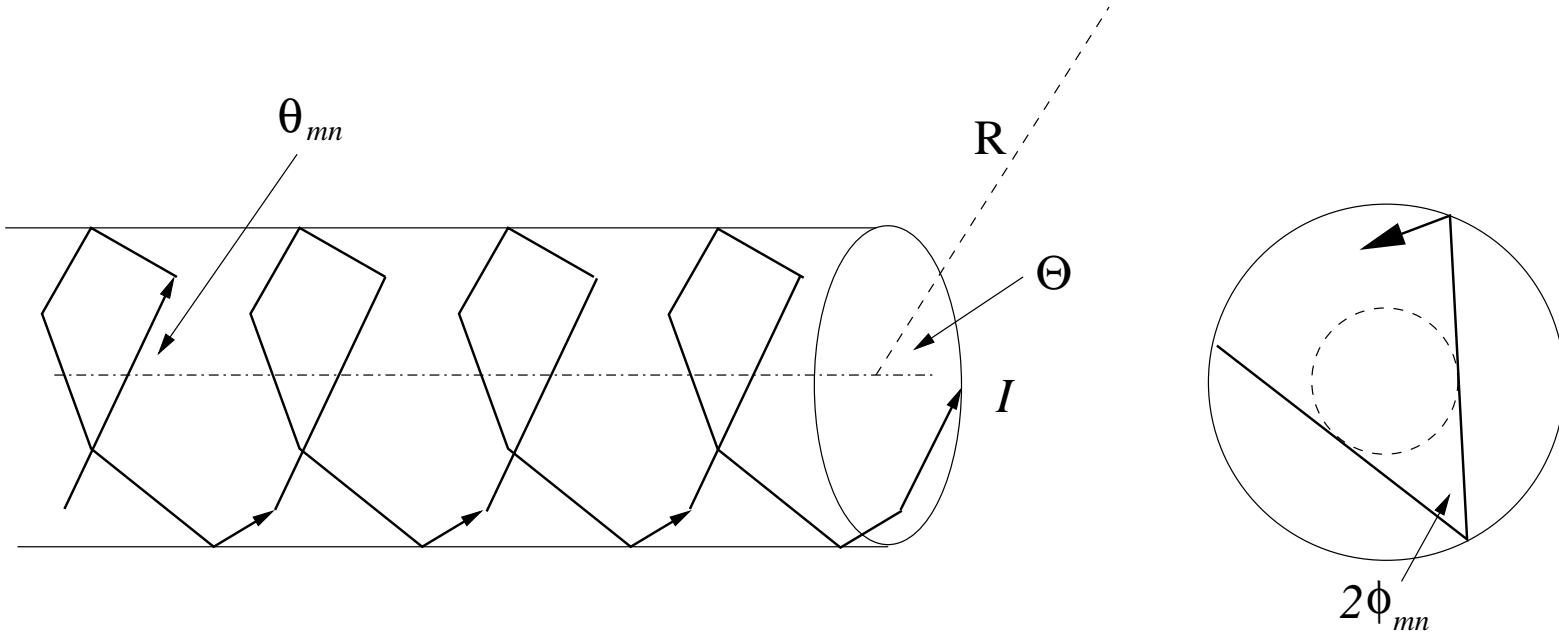
Structure of Acoustic Modes



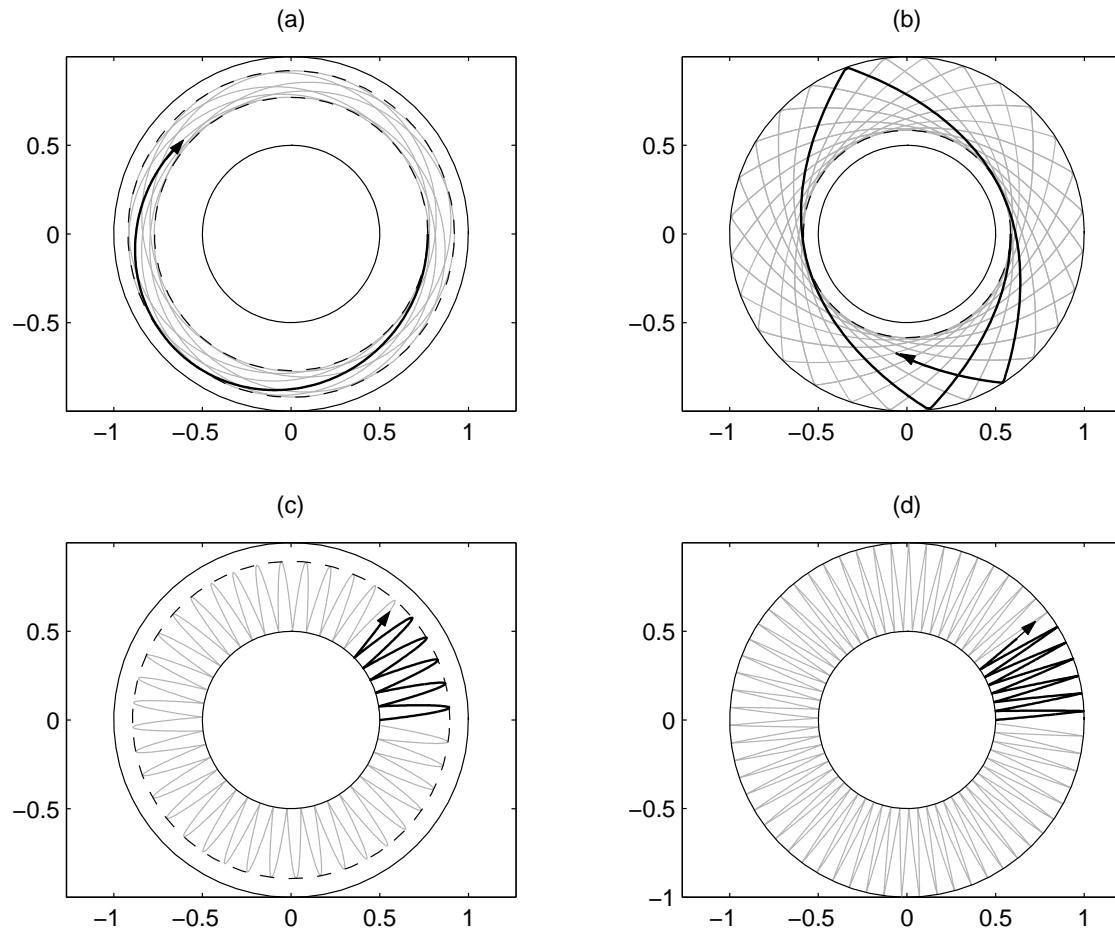
WKB expansion for unsteady pressure satisfies

$$\begin{aligned}\frac{d^2 p}{dr^2} &= -m^2 f(r)p \\ f(r) &\equiv \frac{\Lambda^2}{m^2 c_0^2} - \frac{1}{r^2} - k^2 \\ \Lambda &\equiv \omega - kU - \frac{mW}{r}.\end{aligned}$$

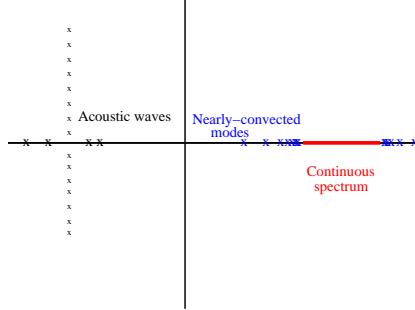
Ray structure in uniform flow



Possible ray structures in swirl.



The continuous spectrum



The continuous spectrum is C_k

$$C_k \equiv \{k : \Lambda(k, r) = \omega - kU(r) - mW(r)/r = 0\}$$

For a given $k \in C_k$, write $\phi = a_0(r - r_c)^\sigma$ near critical layer $r = r_c$, and find

$$\begin{aligned}\sigma &= -\frac{1}{2} \pm \sqrt{\frac{1}{4} - A(r)} \\ A(r) &= \frac{2Wk_c(k_c(Wr)' - mU')}{r^2(m(W/r)' + U'k_c)^2}\end{aligned}$$

If $W = 0$, no swirl, $\sigma = -1, 0$.

Growth of continuous spectrum

- Initial-value problem, given field at $x = 0$ Fourier transform in x , solve for $\bar{\phi}(r, k)$ and invert:

$$\phi(r, x) = \sum \text{discrete modes} + \int_{C_k} \bar{\phi}(r, k) \exp(\mathrm{i}kx) \mathrm{d}k$$

- at $k = k_c(r)$ have singularity $\bar{\phi}(r, k) \sim (k - k_c)^{\sigma(r)}$
- as $x \rightarrow \infty$,

$$\phi(r, x) \sim x^{-\sigma(r)-1} \exp(\mathrm{i}k_c(r)x)$$

- condition for algebraic growth is $\sigma < -1$, i.e.

$$\frac{2Wk_c}{r^2} (k_c(Wr)' - mU') < 0.$$

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Large blade-number asymptotics

Assume large blade number, so azimuthal order $m \gg 1$.
Unsteady velocities now of form

$$(\mathbf{A}(x, r), \phi(x, r)) \exp(imK(x, r) + im\theta - im\Omega t)$$

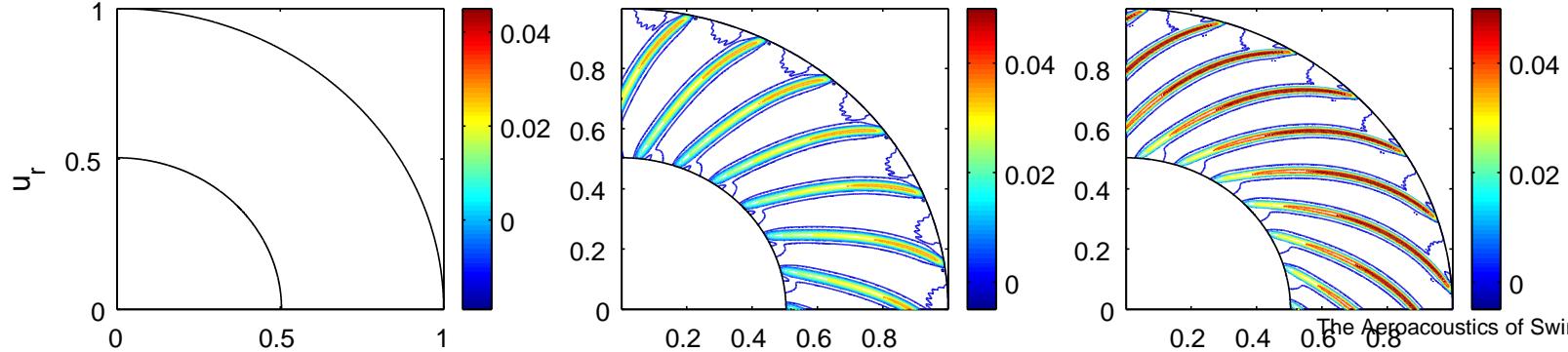
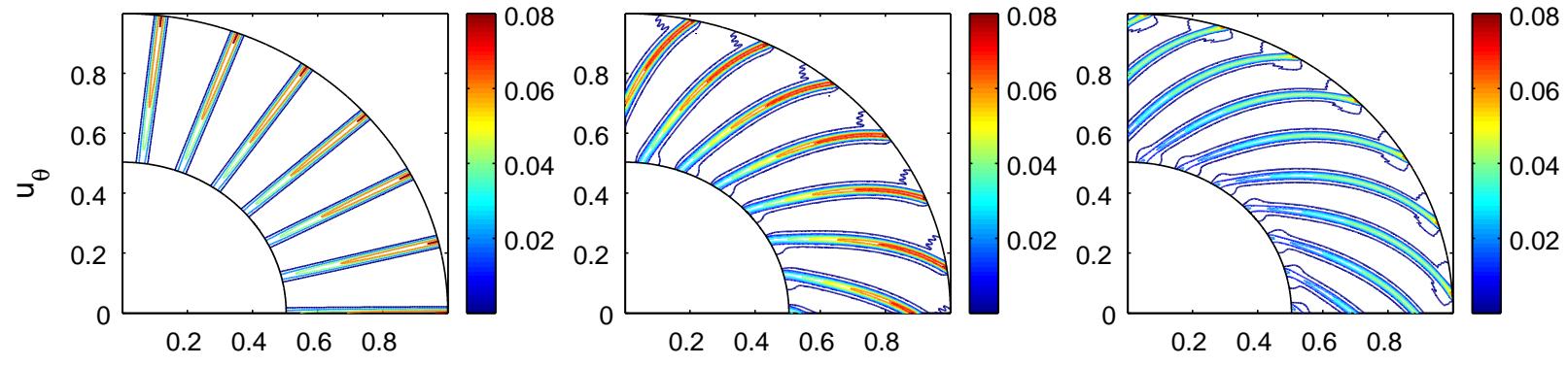
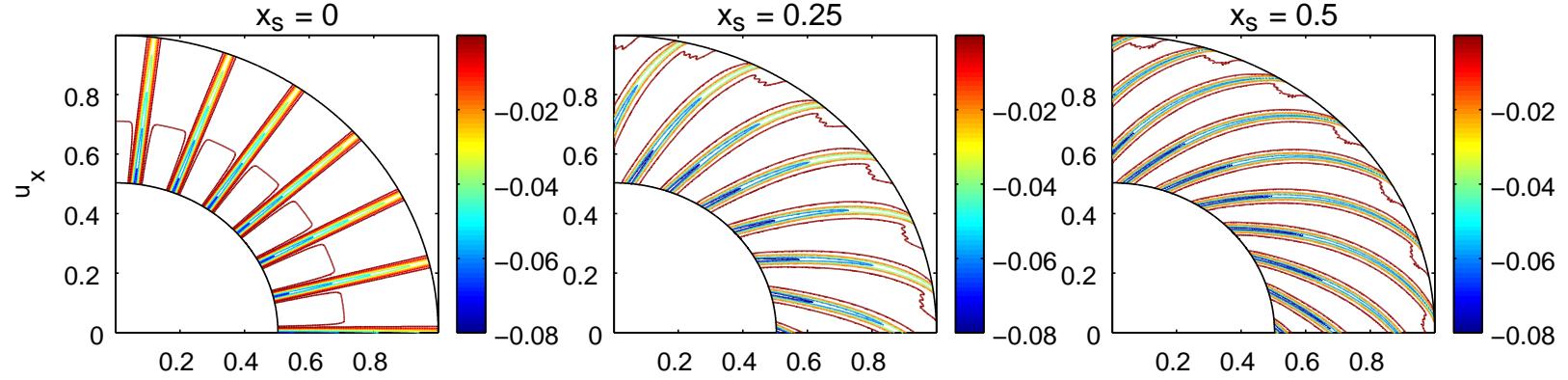
with

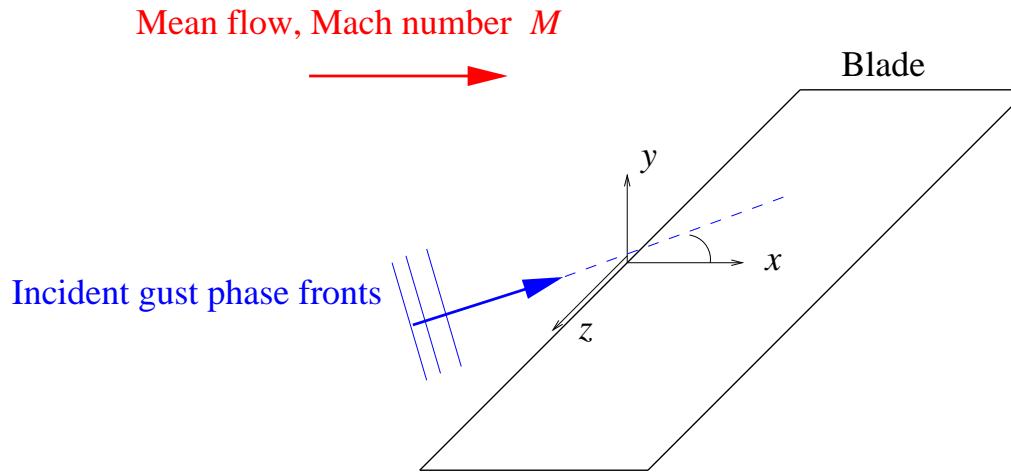
$$-\omega + \frac{mW}{r} + K_x U = 0 ,$$

and

$$\frac{\partial \mathbf{A}}{\partial x} = M(x, r)\mathbf{A} ,$$

and hydrodynamic boundary-layer, width $O(1/m)$, on walls.





Gust component

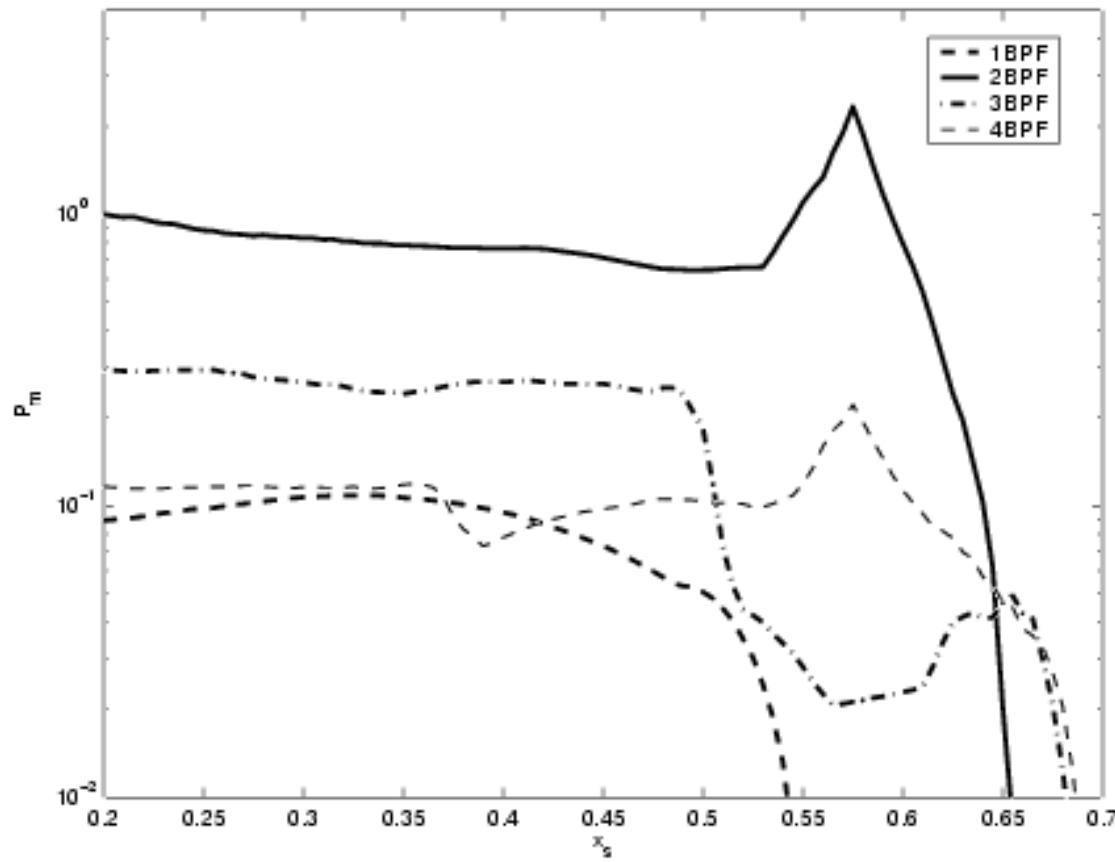
$$\exp(-ikt + ik[x + k_y y + k_z z])$$

Noise radiated if

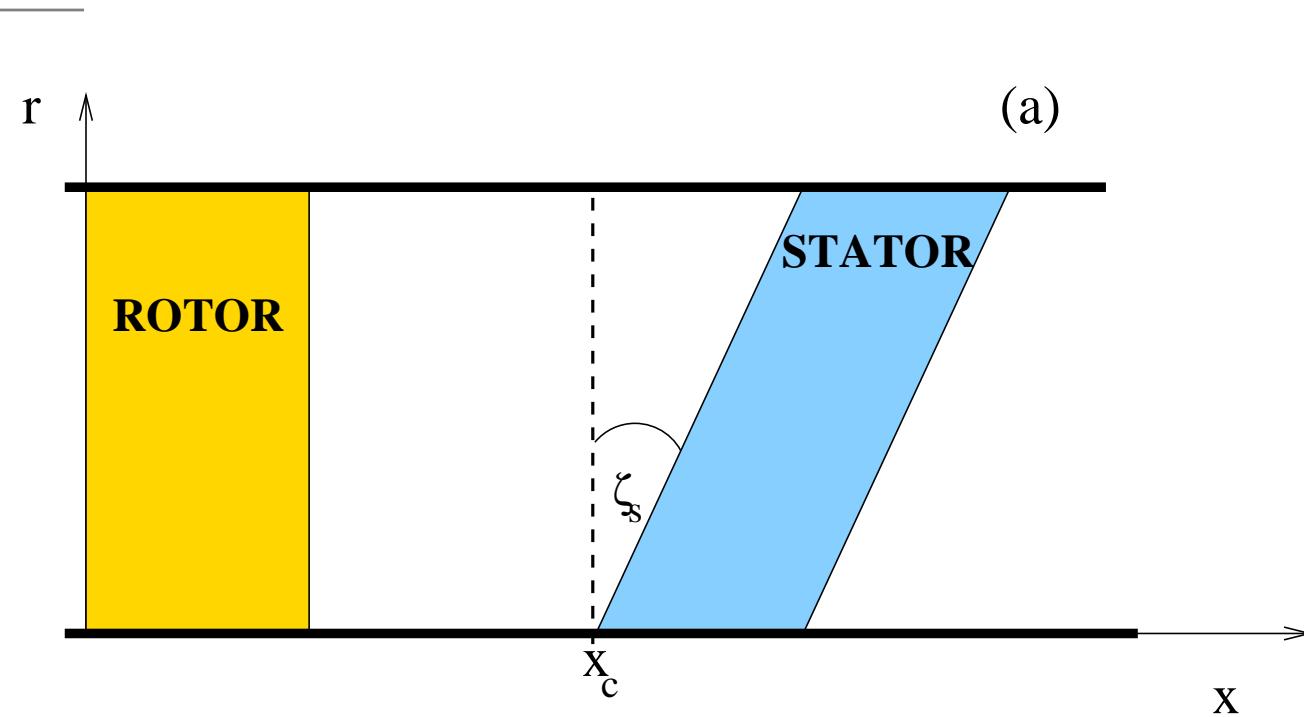
$$|k_z| < M / \sqrt{1 - M^2}$$

i.e. AS LONG AS GUST IS NOT 'TOO' OBLIQUE.

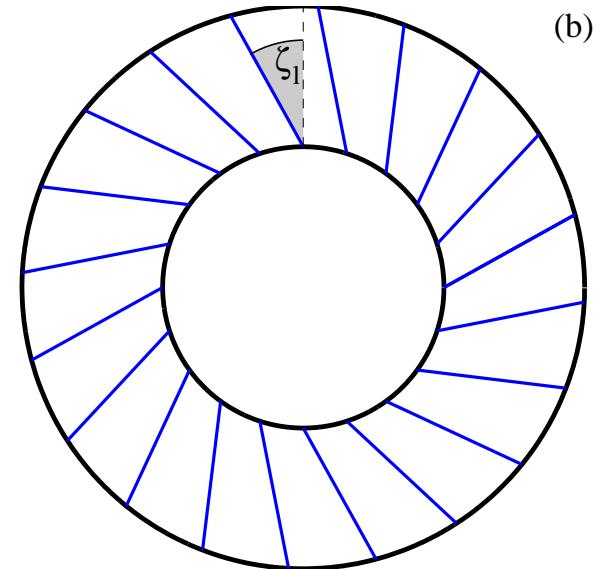
Rotor-stator gap is increased.



Varying blade geometry

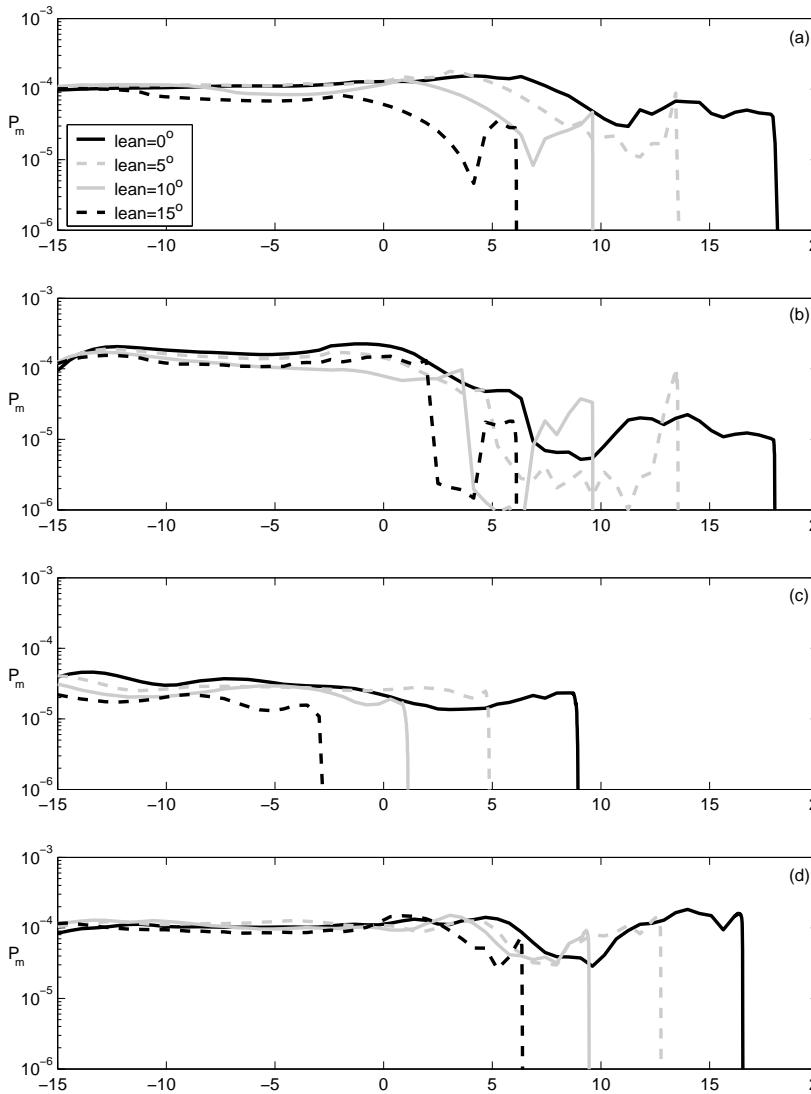


(a)



(b)

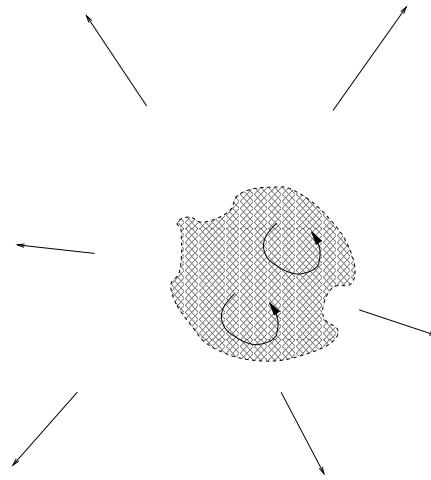
Noise for varying sweep



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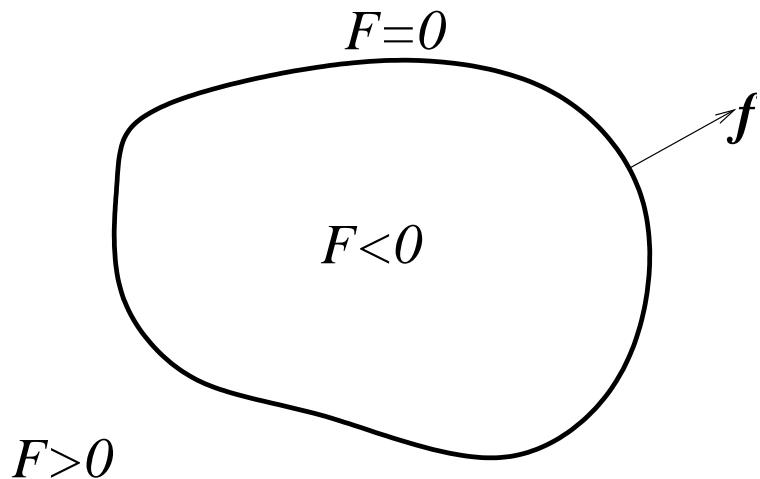
Lighthill's acoustic analogy



$$\left[\frac{\partial^2}{\partial t^2} - c_0^2 \nabla^2 \right] \rho' = \frac{\partial^2}{\partial x_i \partial x_j} [T_{ij}]$$

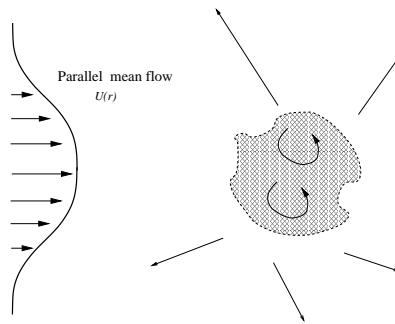
$$T_{ij} = \rho u_i u_j + (p - c^2 \rho) \delta_{ij} + \tau_{ij}$$

Ffowcs Williams' extension



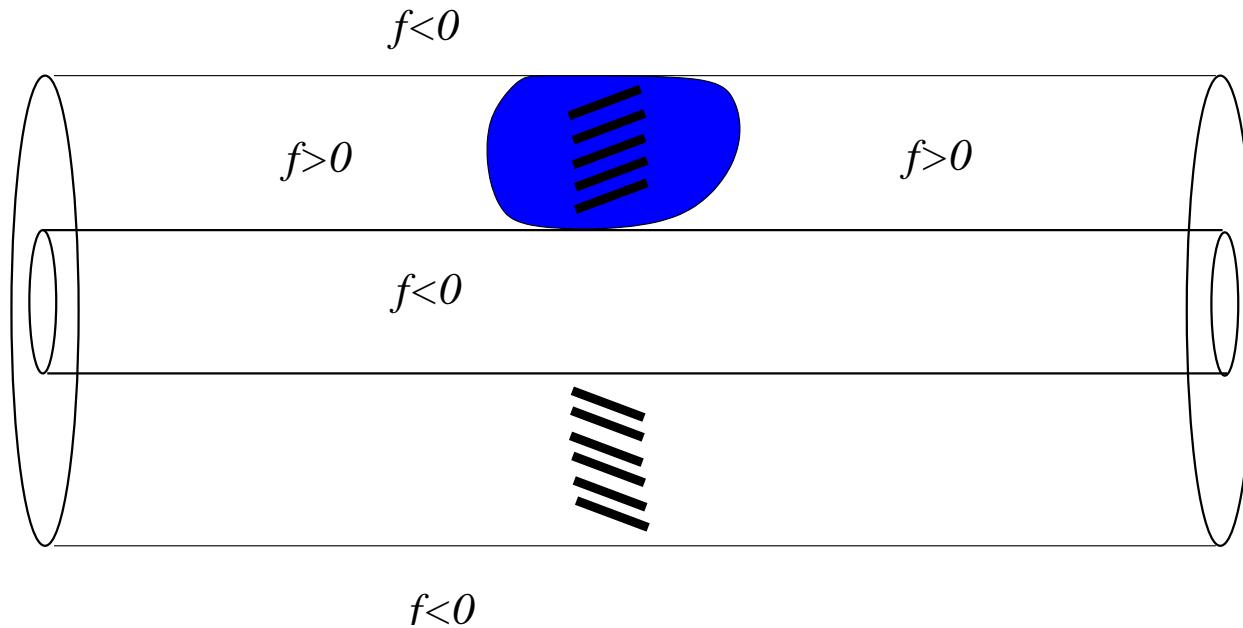
$$\left[\frac{\partial^2}{\partial t^2} - c_0^2 \nabla^2 \right] (\rho' H(F)) = \frac{\partial^2}{\partial x_i \partial x_j} [T_{ij} H(F)] + \frac{\partial}{\partial t} [\rho_0 V_n \delta(F)] - \frac{\partial}{\partial x_i} [f_i \delta(F)]$$

Lilley's equation



Third order partial differential equation in space and time for the pressure fluctuation p :

$$\begin{aligned} \frac{D}{Dt} \left(\frac{1}{c_0^2} \frac{D^2 p}{Dt^2} - \nabla^2 p \right) + 2 \frac{dU}{dr} \frac{\partial^2 p}{\partial r \partial x} &= \\ \frac{D}{Dt} \left(\frac{\partial^2}{\partial x_i \partial x_j} [T_{ij}] \right) - 2 \frac{dU}{dr} \frac{\partial (\nabla \cdot (\rho \mathbf{u} u_r))}{\partial x} \end{aligned}$$



Fluid volume $\mathcal{V}(t)$, surface $\Sigma(t)$, $f > 0$ in $\mathcal{V}(t)$.

$$\mathbf{u}_{to} = \mathbf{U} + \mathbf{u} , \quad \rho_{to} = \rho_0 + \rho , \quad p_{to} = P_0 + p .$$

Mean flow $\mathbf{U} = U_\theta(r)\mathbf{e}_\theta + U_x(r)\mathbf{e}_x$.

Rearrange 4 compressible Navier-Stokes in form

$$\mathcal{L}(\mathbf{H}(f)\mathbf{u}, \mathbf{H}(f)\rho, \mathbf{H}(f)p) = \begin{pmatrix} S_\rho \\ \mathbf{S} \end{pmatrix} \mathbf{H}(f) + \begin{pmatrix} S_{FWH,\rho} \\ \mathbf{S}_{FWH} \end{pmatrix} \delta(f).$$

- \mathcal{L} is linearised Euler operator based on mean swirling flow,
- \mathbf{S} and S_ρ volume sources including quadratic perturbation terms, viscous and non isentropic terms.
- Surface sources

$$\mathbf{S}_{FWH} = \rho_{to} \mathbf{u} \left(\mathbf{u}_{to} - \mathbf{V}^\Sigma \right) \cdot \mathbf{n} + p \mathbf{n} - \bar{\bar{\tau}}_{to} \cdot \mathbf{n}$$

$$S_{FWH,\rho} = \rho_{to} \left(\mathbf{u}_{to} - \mathbf{V}^\Sigma \right) \cdot \mathbf{n} - \rho_0 \left(\mathbf{U} - \mathbf{V}^\Sigma \right) \cdot \mathbf{n}$$

Single equation for p

After a considerable amount of algebra, we find:

$$\mathcal{F}(p) = \mathbb{S} + \mathbb{S}_{FWH},$$

with

$$\begin{aligned}\mathcal{F}(p) &= \mathcal{D} \left(\frac{\mathrm{D}_0}{\mathrm{D}t} \left(\frac{\partial}{\partial r} (\mathcal{M}(p)) \right) \right) + \left\{ \left[\frac{1}{r} \frac{\mathrm{D}_0}{\mathrm{D}t} - \frac{\mathrm{d}(rU_\theta)}{r \mathrm{d}r} \frac{\partial}{r \partial \theta} - \frac{\mathrm{d}U_x}{\mathrm{d}r} \frac{\partial}{\partial x} \right] \mathcal{D} \right. \\ &\quad \left. + 2 \frac{\mathrm{D}_0^2}{\mathrm{D}t^2} \left[\frac{\mathrm{d}U_x}{\mathrm{d}r} \frac{\partial}{\partial x} + \frac{\mathrm{d}}{\mathrm{d}r} \left(\frac{U_\theta}{r} \right) \frac{\partial}{\partial \theta} \right] + \frac{\mathrm{d}}{\mathrm{d}r} \left[\frac{2U_\theta}{r^2} \frac{\mathrm{d}(rU_\theta)}{\mathrm{d}r} \right] \frac{\mathrm{D}_0}{\mathrm{D}t} \right\} \mathcal{M}(p) \\ &\quad - \left[\frac{\partial^2}{r^2 \partial \theta^2} + \frac{\partial^2}{\partial x^2} - \frac{1}{c_0^2} \frac{\mathrm{D}_0^2}{\mathrm{D}t^2} \right] \mathcal{D}^2(p),\end{aligned}$$

with

$$\mathcal{M}(p) = \frac{\mathrm{D}_0}{\mathrm{D}t} \left(\frac{\partial p}{\partial r} \right) + 2 \frac{U_\theta}{r^2} \frac{\partial p}{\partial \theta} - \frac{U_\theta^2}{r c_0^2} \frac{\mathrm{D}_0 p}{\mathrm{D}t} \quad \text{and} \quad \mathcal{D}(u) = - \frac{\mathrm{D}_0^2 u}{\mathrm{D}t^2} - 2 \frac{U_\theta}{r^2} \frac{\mathrm{d}(r U_\theta)}{\mathrm{d}r} u.$$

Zero swirl, no surfaces

Now

$$\mathcal{F}(p) = \frac{D_0^3}{Dt^3} \left[\frac{D_0}{Dt} \left(\frac{1}{c_0^2} \frac{D_0^2 p}{Dt^2} - \nabla^2 p \right) + 2 \frac{dU_1}{dy_2} \frac{\partial^2 p}{\partial y_1 \partial y_2} \right],$$

and

$$\mathbb{S} = \frac{D_0^3}{Dt^3} \left\{ \frac{D_0}{Dt} \left[\frac{\partial^2 \rho_{to} u_i u_j}{\partial y_i \partial y_j} \right] - 2 \frac{dU_1}{dy_2} \frac{\partial^2 (\rho_{to} u_2 u_i)}{\partial y_1 \partial y_i} \right\}.$$

so Lilley's equation is recovered

$$\begin{aligned} & \frac{D_0^3}{Dt^3} \left\{ \frac{D_0}{Dt} \left(\frac{1}{c_0^2} \frac{D_0^2 p}{Dt^2} - \nabla^2 p \right) + 2 \frac{dU_1}{dy_2} \frac{\partial^2 p}{\partial y_1 \partial y_2} \right\} \\ &= \frac{D_0^3}{Dt^3} \left\{ \frac{D_0}{Dt} \left[\frac{\partial^2 \rho_{to} u_i u_j}{\partial y_i \partial y_j} \right] - 2 \frac{dU_1}{dy_2} \frac{\partial^2 (\rho_{to} u_2 u_i)}{\partial y_1 \partial y_i} \right\}. \end{aligned}$$

Integral form

$$p(\mathbf{x}, t) = \int \iiint_V G(\mathbf{x}, t | \mathbf{x}_0, t_0) (\mathbb{S}(\mathbf{x}_0, t_0) + \mathbb{S}_{FWH}(\mathbf{x}_0, t_0)) r_0 d\theta_0 dr_0 dx_0 dt_0 ,$$

$G(\mathbf{x}, t | \mathbf{x}_0, t_0)$ Green's function in duct with swirl.

Move derivatives onto G :

$$p(\mathbf{x}, t) = T_V(\mathbf{x}, t) + T_S(\mathbf{x}, t)$$

with e.g.

$$\begin{aligned} T_S(\mathbf{x}, t) = & + \int \iint_{\Sigma_B(t_0)} \mathbf{S}_{FWH} \cdot \nabla (\mathcal{D}_0^2(G)) - S_{FWH,\rho} \frac{D_0}{Dt_0} (\mathcal{D}_0^2(G)) d\Sigma_0(t_0) dt_0 \\ & + \int \iint_{\Sigma_B(t_0)} 2S_{FWH,\theta} \frac{U_\theta}{r_0} \mathcal{R}_{0,1}(G) d\Sigma_0(t_0) dt_0 + \int \iint_{\Sigma_B(t_0)} S_{FWH,r} \mathcal{R}_{0,2}(G) d\Sigma_0(t_0) dt_0 \end{aligned}$$

Test case

Simple blades:

$$p(\mathbf{x}, t) = \iiint_{\bigcup_j \Sigma_{B,j}(t_0)} p(\mathbf{x}_0, t_0) \mathcal{T}_0(G(\mathbf{x}, t | \mathbf{x}_0, t_0)) d\Sigma_0(t_0) dt_0 ,$$

with

$$\mathcal{T}_0(G) = \left[n_{x,j} \mathcal{D}_0^2 \frac{\partial G}{\partial x_0} + n_{\theta,j} \left(\frac{\mathcal{D}_0^2}{r_0} \frac{\partial G}{\partial \theta_0} + 2 \frac{U_\theta}{r_0} \mathcal{R}_{0,1}(G) \right) \right] .$$

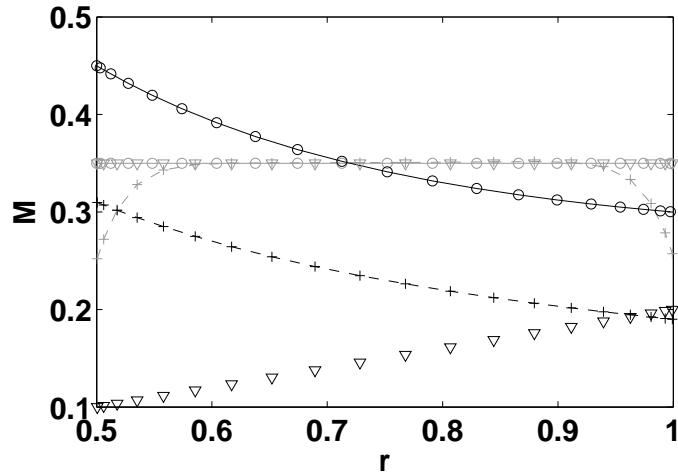
Rotor steady distortion:

$$\mathbf{u}_{inc}(\mathbf{x}) = U_x^\infty \left(1 - \alpha e^{-\ln 2 \left(\frac{\theta}{\theta_0} \right)^2} \right) \mathbf{e}_x = \sum_{q \in \mathbb{Z}} u_q e^{i q \theta} \mathbf{e}_x$$

Blade forcing: flat-plate response theory.

Parameters

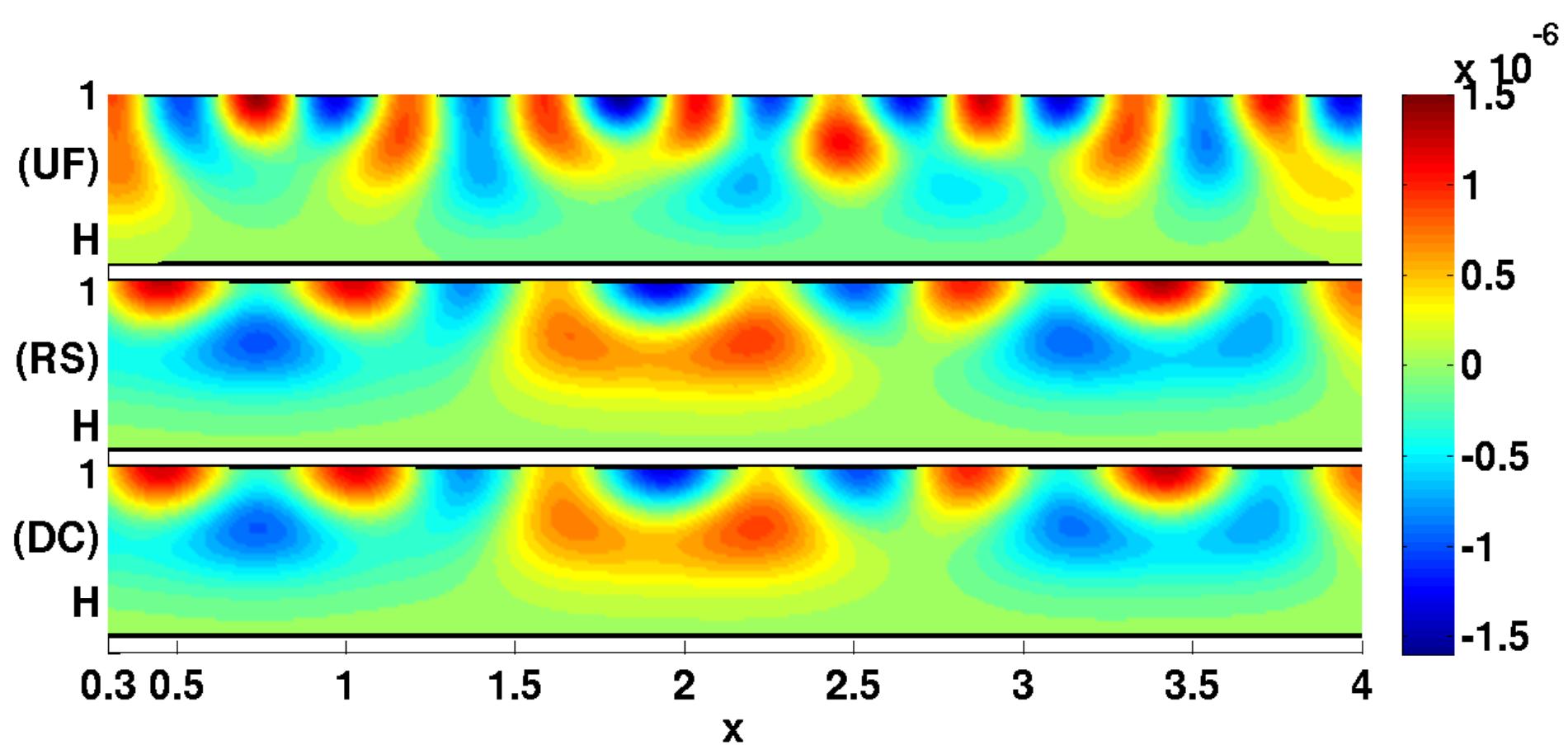
Inspired by 22-in NASA fan SDT, Hughes et al 2002.



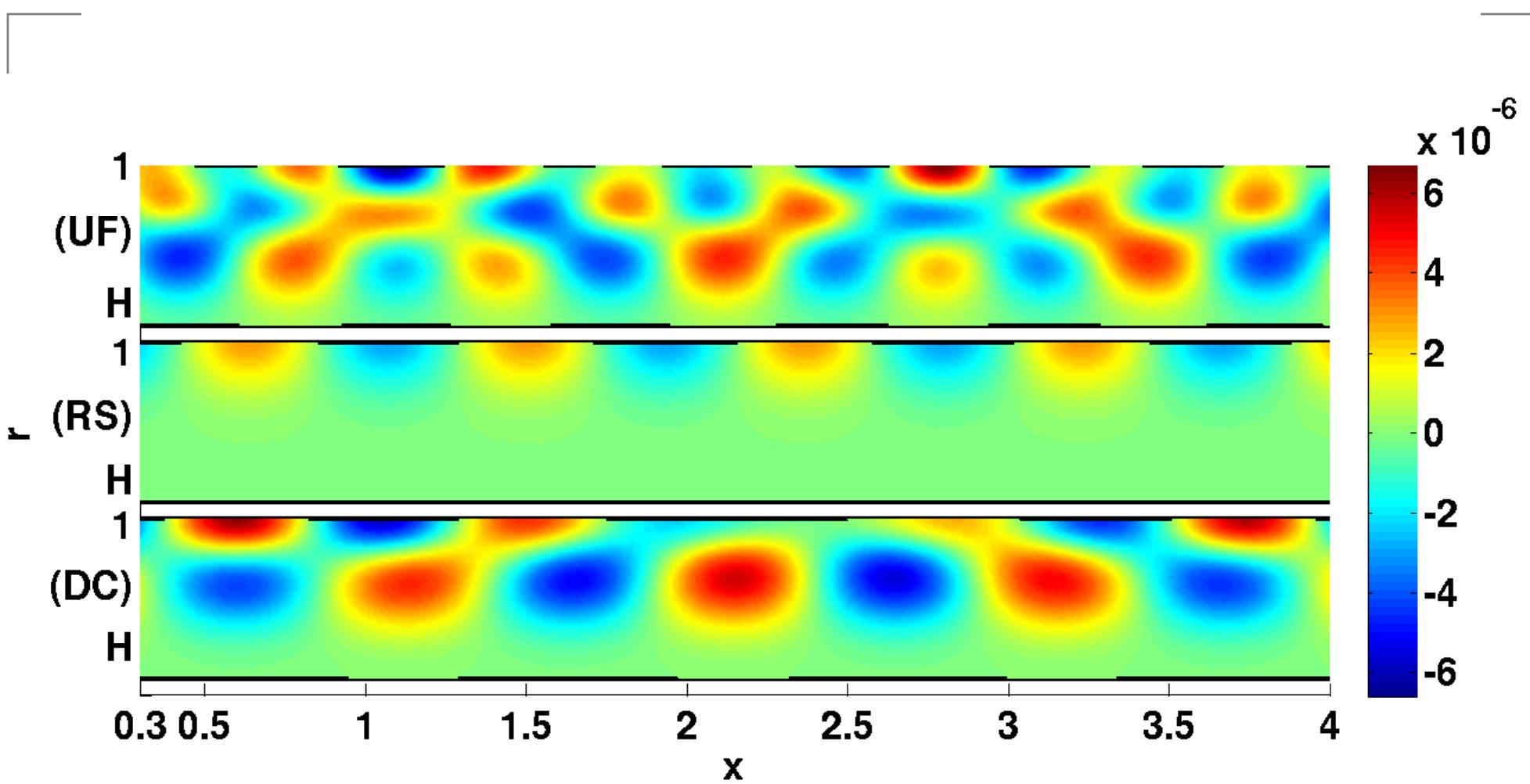
+ symbol realistic swirl

▽ symbol solid-body swirl plus uniform axial flow

Solid body swirl and uniform axial



Realistic mean flow



Ongoing work

- Rotor broadband self noise in swirling flow.
- Non isentropic swirling flow turbine noise.